Amendments to the Specification:

Please replace paragraph **[0007]** with the following rewritten paragraph:

[0007] In the arrangement disclosed in JP-A-H09-86749, during absence of the paper sheet P between the rollers 40, 41, the radially outer end 45 of the driven roller 41 is positioned within the annular groove 43 of the drive roller 40, without the radially outer end 45 being in contact with a bottom surface of the groove 43. In this instance, since the rollers 40, 41 are not in contact with the each other, the flexible shaft 42 holding the driven roller 41 does not receive any load, or is slightly ~~deflexed~~ flexed downwardly due to weight of the driven roller 41. During presence of the paper sheet between the rollers 40, 41, the flexible shaft 42 is ~~deflexed~~ flexed upwardly as a result of the upward displacement of the driven roller 41. In this instance, a restoring force or spring load generated by the ~~deflexed~~ flexed flexible shaft 42 acts on the driven roller 41, whereby driven roller 41 is ~~force~~ forced onto the paper sheet P.

Please replace paragraph **[0008]** with the following rewritten paragraph:

[0008] Therefore, in the arrangement of JP-A-H09-86749, for reliably and accurately feeding the paper sheet P, an amount of the intersection or overlap of the radially outer end portions of the respective rollers 40, 41 has to be sufficiently large, so that the flexible shaft 42 is upwardly ~~deflexed~~ flexed by an amount required for obtaining the predetermined amount of pressing force during presence of the paper sheet P between the rollers 40, 41. It is noted that the intersection or overlap amount can be represented by a distance L1 (as indicated in Fig. 12) between the radially outer end 45 of the driven roller 41 and the radially outer end 44 of the drive roller 40.

Please replace paragraph **[0035]** with the following rewritten paragraph:

[0035] Each of the second driven rollers 41 includes a radially inner core portion 41a and a gear-like or toothed radially outer end portion 49 which has a plurality of radially-

extending projections, as shown in Fig. 6. The second driven rollers 41 are rotatably held independently of each other, by respective flexible shafts 42 (each provided by a coil spring shaft) which are arranged in series along a straight line, as shown in Fig.3, and are driven to be rotated following rotation of the second drive roller 40. Each of the second driven rollers 41 is mounted on an axially intermediate portion of a corresponding one of the flexible shafts 42, as shown in Fig. 4A. Each of the flexible shafts 42 is supported by a supporting device in the form of a pair of inside support members 47 and a pair of outside support members 48. The inside and outside support members 47, 48 are provided by respective plate members extending from a holder plate (not shown) which is arranged to be opposed to the second drive roller 40 and which is a part of the main body 2 or is connected to the main body 2. The outside support members 48 have respective distal end surfaces 48a, as shown in Fig. 4B, which are to be brought into contact with axially opposite end portions of the flexible shafts 42. That is, the outside support members 48 serve as a displacement limiter for limiting displacement of the axially opposite end portions of the flexible shaft 42 in a direction away from the drive roller 40, i.e., in the rightward direction as seen in Fig. 4B. Each of the inside support members 47 is located between the driven roller 41 and a corresponding one of the outside support members 48, as shown in Fig. 4A. Each inside support member 47 is divided into two parts by an elongated hole or slot 47a which is formed in its widthwise central portion and extends in its longitudinal direction, and has an engaging portion 47b, in its distal end portion, which projects from one of the above-described divided two parts toward the other part, as shown in Fig. 4B. The axially intermediate portion of the flexible shaft 42 is accommodated in the elongated holes 47a of the inside support members 47. The driven roller 41 and the axially intermediate portion of the flexible shaft 42 are allowed, owing to the elongated holes 47a of the inside support members 47, to be displaced in the direction away from the drive roller 40. However, their displacement toward the drive roller 40 is limited by

the engaging portions 47b of the inside support members 47. The driven rollers 41 are positioned relative to the drive roller 40 such that each of the driven rollers 41 is opposed to a corresponding one of the annular grooves 43 of the drive roller 40 and such that a radially outer end 45 of each driven roller 41 is positioned within the corresponding annular groove 43 of the drive roller 40. That is, the toothed radially outer end portion 49 of each driven roller 41 overlaps with a radially outer end portion of the drive roller 40. In the present embodiment, each driven roller 41 is made of a resin such as polyacetal (POM) which preferably contains a waxy opaque material sold under the trademark TEFLON. ~~Teflon (registered trademark).~~ However, the entirety of each driven roller 41 or only the toothed radially outer end portion 49 may be made of a metallic material such as stainless steel (SUS). Further, while the toothed radially outer end portion 49 is provided by two thin plates which are held in parallel with each other in the present embodiment, the radially outer end portion 49 may be provided by a single plate.

Please replace paragraph **[0037]** with the following rewritten paragraph:

[0037] Each flexible shaft 42 supporting the corresponding driven roller 41 is fixed by the above-described inside and outside support members 47, 48 in a position relative to the drive roller 40 such that the overlap amount corresponds to a distance L1 (i.e., the same distance as in the above-described conventional feeding device) if the tongue member 46 as the overlap-amount limiter were absent in the present feeding device 10. This distance L1, larger than the above-described distance L2, is determined on the basis of an amount of spring load required for obtaining a predetermined amount of pressing force. In absence of the tongue member 46, the radially outer end portion 49 of each driven roller 41 overlaps with the radially outer end portion of the drive roller 40 by the distance L1, and the flexible shaft 42 holding the driven roller 41 does not receive any load, or is slightly ~~deflexed~~ flexed downwardly due to weight of the driven roller 41. It is noted that each of the annular grooves

43 of the drive roller 40 has a radial depth sufficiently large such that the driven roller 41 is not brought into contact with a bottom surface of each annular groove 43 of the drive roller 41 even in absence of the tongue member 46.

Please replace paragraph **[0038]** with the following rewritten paragraph:

[0038] The overlap amount is reduced to correspond to the distance L2, by the provision of the tongue member 46 as the overlap-amount limiter which is brought into contact with the radially outer end 45 of the driven roller 41. As a result of the reduction of the overlap amount, the flexible shaft 42 on which the driven roller 41 is mounted is ~~deflexed~~ flexed upwardly, as shown in Fig. 5, since the driven roller 41 together with the axially intermediate portion of the flexible shaft 42 is raised by the tongue member 46 against the elastic force of the flexible shaft 42, by an amount corresponding to a difference between the distances L1 and L2.

Please replace paragraph **[0043]** with the following rewritten paragraph:

[0043] There will be described a specific example of the arrangement of the second drive and driven rollers 40, 41. In this specific example of the arrangement, the driven rollers 41 are positioned relative to the drive roller 40 such that the distance L1 (i.e., the overlap amount in absence of the tongue member 46 as the overlap-amount limiter) is 1.0 mm, while the tongue member 46 is positioned relative to the driven rollers 41 such that the distance L2 (i.e., the overlap amount in presence of the tongue member 46 which upwardly forces the driven rollers 41) is 0.3 mm. In this arrangement, the driven rollers 41 are upwardly forced by the tongue member 46, the driven rollers 41 receive a spring load of 14 gF (=14 x 9.8 x 10^{-3}N). When the paper sheet P is positioned between the drive and driven rollers 40, 41, as shown in Fig. 7, the paper sheet P is gripped by and between the rollers 40, 41 with a pressing force (spring load) of 20 gF. This means that, for enabling the paper sheet P to be gripped with the pressing force of 20 gF, the paper sheet P has to raise each driven roller 41 by an

amount as large as 1.0 mm upon its entrance between the rollers 40, 41 in the conventional feeding device, as shown in Fig. 11, in which the tongue member 46 as the overlap-amount limiter is absent. In the present feeding device 10 equipped with tongue member 46, the amount by which each driven roller 41 has to be raised by the paper sheet P is as small as 0.3 mm, since each driven roller 41 is already raised by the tongue member 46 before the paper sheet P enters between the rollers 40, 41.